

GAS FLOW CONTROL

Field of the Invention

The invention relates to gas flow control, and more specifically, to a gas flow controller.

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Background

Many types of gas flow controllers have been implemented for a variety of applications.

For example, many known control arrangements for gas burners include a main valve, a servo valve and a servo controller, wherein the servo controller serves to control a gas

10 output pressure by controlling the opening of the main valve.

For modulating the gas outlet pressure, an actuator is provided for the servo valve and achieves an opening and closing of the servo valve at the pertinent frequency, which can be modulated.

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Fig. 2 shows one such gas flow controller to which various implementations of the present invention may be applicable. This controller comprises a main valve, the valve head 21 of which cooperates with a valve seat 22 and is loaded by a spring 30 into its closing position.

The valve head 21 is connected to a diaphragm 23 above which a first gas chamber 24 is

20 provided. The gas flows from the inlet 34 into the second gas chamber 25 and can flow on into the third gas chamber 31 when the main valve is open, and from there to the outlet 35.

If the main valve is closed, a gas flow is prohibited. In the closing position, the two-way

valve 29, which is operable electrically, is in its closed position. The two-way valve 29 is held in this position due to an appropriate pressure difference in the gas chambers 24, 25

25 acting on the diaphragm, and due to the force of the spring 30 acting on the valve seat 21.

Thus, gas flows into the first gas pipe 27 through the second gas pipe 26 which has a

certain resistance adapted, if necessary, to be influenced by a throttle 32, since a flow through the two-way valve 29 is not possible due to the closed position of the same.

If there is a pressure equalization between the first gas chamber 24 and the second gas chamber 25, the main valve is nevertheless securely held in its closed position, since it is loaded into the closed position by the pressure spring 30. If the two-way valve 29 is opened, gas flows via the first gas pipe 27 and the second gas pipe 26 through the two-way valve 29 into the third gas pipe 28 ending in the third gas chamber 31. The pipe portion upstream of the two-way valve 29 and through the two-way valve 29 shows a certain flow resistance that can, if necessary, be influenced by a throttle 33. Thus, gas flows out of the first gas chamber 24 until the gas pressures in the first gas chamber 24 and in the third gas chamber 31 are equal. Furthermore, no additional gas flows via the first and second gas pipes 27, 26 into the first gas chamber 24, as this rather flows via the two-way valve 29 into the third gas pipe and, thus, into the third gas chamber 31.

Accordingly, if the two-way valve 29 is shut again, *i.e.*, if a flowing-off via the third gas pipe 28 into the third gas chamber 31 is impossible, there is another pressure equalization between the first gas chamber 24 and the second gas chamber 25, with the result that the main valve closes again. It is obvious that, for the mode of operation of this control means, the coordination of the pipe resistances is of major importance. Such control means must have a certain minimum closing speed. Thus, to allow an appropriately quick closing of the main valve, the pipe resistances have to be coordinated accordingly. Furthermore, a certain opening speed has to be ensured and, of course, a certain maximum opening of the main valve, ensuring the desired gas flow from the inlet 34 to the outlet 35. No pressure modulation by mere adjustment of a certain differential pressure between the first gas chamber 24 and the second gas chamber 25 is possible with this controller. The cross-sections and flow resistances of the gas pipes are coordinated such that the required and desired opening and closing speeds are reached. Thus, when the two-way valve 29 is opened, the main valve is automatically taken into its open position at a predetermined speed, whereas, when the two-way valve 29 is closed, a complete closing of the main valve sets in along with the desired closing speed. Thus, a pressure modulation is only possible if the servo two-way valve 29 is operated in a pulsed manner with the pulse width being modulatable.

Summary

The present invention is directed to a control arrangement for gas flow that addresses
5 challenges including those discussed above.

According to an example embodiment of the present invention, a gas flow controller
comprises a main valve and a servo valve adapted for effecting differential pressure
modulation therebetween without necessarily modulating the actuation frequency of the
10 servo valve.

According to a more particular example embodiment of the present invention, a gas flow
controller includes a three-way servo valve means connected via gas lines for selectively
controlling differential pressure across a diaphragm for controlling a main valve. The
15 three-way valve is connected to a first gas chamber that is substantially formed so as to be
closed off, and via the pressure of which the main valve is operated. Furthermore, the three-
way valve means is connected to a second gas chamber in the inlet area of the gas flow
controller, and to a third gas chamber in the outlet area of the control valve. The diaphragm
separates the first and second gas chambers and is responsive to differential pressures
20 therebetween. The three-way valve means can be provided by a single three-way valve, or
by a suitable combination of two-way valves.

In one implementation, the three-way valve means is adapted for switching such that either
the first gas chamber is connected to the second gas chamber or the first gas chamber to the
25 third gas chamber. Via such a connection, the main valve can be easily controlled,
addressing challenges including those discussed above. Moreover, the opening cross-
section of the main valve can be modulated for achieving the desired modulation of the gas
flow by the gas flow controller.

The above-discussed arrangement can be implemented in a variety of manners. For instance, in one implementation, the main valve is put into its closed position by a suitable pressure difference between the gas chambers and, furthermore, by means of a spring. The main valve is then opened by negative pressure in the first gas chamber vis-à-vis the second

5 gas chamber. For this, the main valve is in operative connection with a diaphragm dividing off the first gas chamber from the second gas chamber. The negative pressure in the first gas chamber facilitates the actuation of the diaphragm in a direction toward the first chamber (and the lower pressure), thus moving the valve in the same (open) direction.

10 In another example embodiment of the present invention, opening and closing speeds of the main valve are selected to achieve a variety of results. To meet these, the cross sections and flow resistance in the gas pipes and through the three-way valve means are adjusted to the desired opening and closing speeds of the main valve. In a more particular implementation, the cross section and the flow resistance in the gas pipes, the three-way valve means,

15 particularly in the gas pipe which connects the three-way valve means with the second gas chamber, and the corresponding inlet area of the three-way valve are adjusted for a modulation of the opening of the main valve. A modulation that is effective and well controllable is readily effected when the desired opening cross section of the main valve is reproducible and reliably adjustable. With the approaches discussed herein, such

20 modulation is achieved, and the gas pipe for the pressure build-up in the first gas chamber can be designed in a completely independent manner from the gas pipe for pressure release in the first gas chamber.

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Brief Description of the Figures

In the following, the invention will be described more precisely by referring to example embodiments in combination with the attached drawings wherein:

30 Fig. 1 shows a principle sketch of a gas flow controller, according to an example embodiment of the invention; and

FIG. 2 shows a valve arrangement to which various example embodiments of the present invention are applicable.

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Detailed Description

In the following description, reference is made to the accompanying drawings that form a part hereof, and in which is shown by way of illustration particular embodiments in which the invention may be practiced. It is to be understood that other embodiments may be utilized, as structural and operational changes may be made without departing from the 10 scope of the present invention.

According to an example embodiment of the present invention, a gas flow controller includes a control valve arrangement coupled to gas chambers on opposite sides of a diaphragm and arranged for controlling a main valve as a function of differential pressure 15 across the diaphragm. The control valve is coupled to an inlet and outlet of the gas flow controller, and to an auxiliary chamber limited by the diaphragm. The inlet and outlet are in a chamber separated by the main valve, and are coupled when the main valve is open. The control valve arrangement controls the differential pressure (between the inlet and the auxiliary chamber) by selectively coupling the inlet, outlet and auxiliary chambers.

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FIG. 1 shows a gas flow controller, according to another example embodiment of the present invention. Gas flows from the inlet 14 to the second gas chamber 5. If the main valve is in the shown closed position, *i.e.*, if the valve head 1 rests on the valve seat 2, the gas cannot flow on to the third gas chamber 11 and, thus, to the outlet 15. The gas flow is 25 interrupted and the controller is locked. Thus, the valve head 1 is pressed into its closed position, thus onto the valve seat 2, by means of a pressure spring 10. The valve head 1 is in operative connection with a diaphragm 3, which cuts off a first gas chamber 4 in the upper area of the gas flow controller. The three-way valve 9 communicates with three gas pipes, the first gas pipe 7 of which communicates with the first gas chamber 4 above the 30 diaphragm 3. The second gas pipe 6 connects the three-way valve 9 to the second gas

chamber 5. The third gas pipe 8 finally connects the three-way valve 9 to the third gas chamber 11.

In the closed position shown, the three-way valve is switched such that the first gas pipe 7
5 and the second gas pipe 6 communicate with each other. Due to this circuit, there is a pressure difference between the first gas chamber 4 and the second gas chamber 5, which holds the valve head 1 of the main valve in the closed position together with the force of the spring 10.

10 For opening the main valve, the three-way valve 9 is taken into a position in which it connects the first gas pipe 7 to the third gas pipe 8. This communication serves to relieve the pressure in the first gas chamber 4 towards the third gas chamber 11 and, thus, towards the outlet side of the gas flow controller. Due to the flow resistance upstream of the three-way valve 9, the pressure drops upstream of the three-way valve 9, so that the pressure in
15 the first gas chamber 4 is speedily reduced. Due to the pressure difference produced in this way between the first gas chamber 4 and the second gas chamber 5, the main valve is opened, since the pressure in the second gas chamber 5 becomes higher than the pressure in the first gas chamber 4. Via the diaphragm 3, the valve head 1 is, thus, moved upwards against the force of the spring 10, and it is lifted off the valve seat 2 thereby releasing the
20 opening cross section of the main valve. If the main valve is to be closed again, the three-way valve 9 is simply taken into the position named in the beginning, in which the first gas pipe 7 communicates with the second gas pipe 6. In this way, pressure is again built up in the first gas chamber 4, and the main valve is closed. A suitable intermediate positioning of the three-way valve 9 makes it possible to adjust a certain pressure difference between the
25 first gas chamber 4 and the second gas chamber 5. For this, the inflow of the gas via the second gas pipe 6 and the first gas pipe 7 into the first gas chamber 4 as well as the outflow of the gas via the third gas pipe 8 have to be adjusted accordingly.

30 The flow cross sections through the gas pipes as well as through the three-way valve 9 are coordinated with the opening and closing behavior of the main valve, just like the flow

resistance. To achieve a quick opening and/or closing, large cross sections and low flow resistance are useful. However, the coordination of the cross sections and the flow resistance also may be effected with respect to the desired modulation behavior of the main valve. In one instance, the communication of the first gas chamber 4 with the second gas chamber 5 is coordinated with the opening behavior of the main valve by selecting the cross section and flow resistance of gas lines coupling the first and second gas chambers. These selections can be made, for example, with regard to characteristics of the spring 10 as well as the diaphragm 3 to achieve the desired modulation of the opening cross-section of the main valve.

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For general information regarding gas valve control and for specific information regarding an example approach to gas valve control that may be implemented in connection with the present invention, reference may be made to German Patent No. DE 100 26 035 A1, which is fully incorporated herein by reference.

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The foregoing description of various embodiments of the invention has been presented for the purposes of illustration and description. It is not intended to be exhaustive or to limit the invention to the precise form disclosed. Many modifications and variations are possible in light of the above teaching. It is intended that the scope of the invention not be limited 20 with this detailed description.